1st EPERC International Conference Pressure Equipment Innovation and Safety Roma, 1-3 April 2019

Nonlinear analysis in PE Design Codes

Recommendations for codified rules improvements

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<u>Rev. 0</u>



1-3 April 2019

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 - Code Convergence Method
 - Short Introduction of the Project
- ✓ Part 1: Existing Code Comparisons
 - Monotonic loads
 - Cyclic loads
- ✓ Part 2: Recommended Practices
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 - Appendix 2: Cyclic plasticity model
- ✓ Part 3: Benchmarks
- ✓ Other topics in the Project
- ✓ Open points
- Conclusion and planning



Code Convergence method...

On selected topics

- 1. Detailed Comparison of existing Codes: nuclear + non nuclear
- Identify: Gaps and Needs
- 3. International review of corresponding report by each Code Org.
- 4. Recommended practice document associated with all proposals validation
- 5. International Benchmarks on "realistic" cases
- 6. Final "Harmonized" Code Case
- 7. Large international participation & review on 4-5-6 by Code Organizations and International Expert Groups

Minimize Future Code Divergence
Facilitate Areas of Convergence

Procedure Successfully applied for:

- Class 1 design rules,
- NDE personal qualification,
- Welding qualification

On-going for: Fatigue Rules

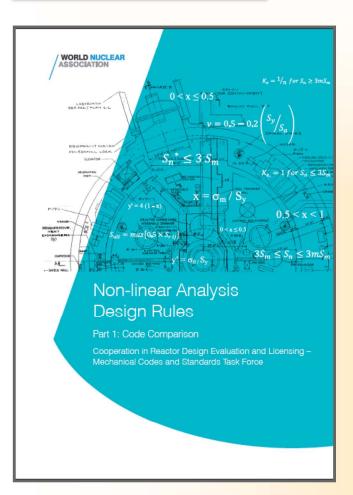
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Short Introduction of the Project

- ✓ Question: How to improve Pressure Equipment Code Design rules considering nonlinear behavior of material?
- Damages Concerned
 - Plastic collapse
 - Plastic instability
 - Local Failure
 - Fracture of cracked PE
 - Fatigue: K_e and cyclic direct evaluation
 - Plastic shakedown and ratcheting
- Other aspects:
 - the stress classification rules : reinforced nozzles
 - Piping analysis: elastic follow-up (later); Dynamic cyclic loads (seismic)
 - Validation of more simple rules, analytical formulae...
- Loads:
 - Mechanical and thermal
 - Quasi-static, cyclic or dynamic
- Analysis methods:
 - > Elastic
 - Elastic-plastic monotonic/cyclic
 - Limit load
- No buckling; no creep (later)



Existing Code comparisons



✓ Report available

- Main conclusions:
 - No available Code answers to all the "questions"; only partly
 - 2 Codes have more information to develop this type of analysis:
 - French AFCEN RCC-MRx
 - USA ASME BPVC Section VIII Div. 2
 - Large improvements of existing Codes is needed
- The corresponding report lists:
 - Major Open Points
 - Major Gaps and Needs



Monotonic loads

	Plastic collapse				Plastic instability				Stress triaxiality	
	Limit Analysis		Direct elastic-plastic analysis		Limit Analysis		Direct elastic-plastic analysis		Direct elastic-plastic analysis	
	Material properties	Criteria	Material properties	Criteria	Material properties	Criteria	Material properties	Criteria	Material properties	Criteria
RCCM	Y	Y	N	N	N	N	N	Y	N	N
ASME III	Υ	Υ	N	N	Υ	Υ	N	N	N	N
JSME	Y	Y	N	N	N	N	N	N	N	N
RCC-MRx	Y	Y	Υ	Υ	Y	Υ	Y	Y	N	N
Russia	N	N	N	N	N	N	N	N	N	N
КТА	N	N	N	N	N	N	N	N	N	N
R5	N	N	N	N	N	N	N	N	N	N
ASME VIII	Y	Υ	Υ	Y	Y	Υ	Υ	Y	Y	Υ
EN 13445	Y	Y	N	N	N	N	N	N	N	N

Y: Yes - N: No - P: Partially



Cyclic Loads

		Plastic sl	nakedown	Fatigue and K _e			
	l I	Direct elastic-	plastic anal	Direct elastic-plastic analysis			
	Material properties	Material constitutive equation	Criteria	Extrapolation rules	Material properties	Material constitutive equation	Criteria
RCCM	N	N	N	N	N	N	N
ASME III	N	N	N	N	N	N	N
JSME	Υ	Р	Y	N	Y	N	Υ
RCC-MRx	Р	Р	N	Y	Υ	Р	N
Russia	N	N	N	N	Υ	Υ	Y
KTA	N	N	N	N	N	N	N
R5	N	N	Υ	N	N	N	N
ASME VIII	Υ	N	Y	N	Υ	N	Υ
EN 13445	N	N	N	N	N	N	N

Y: Yes - N: No - P: Partially





Part 2: Recommended practices

EPERC New Draft Report (under preparation):

- √ 1. References
- ✓ 2. Introduction
- ✓ 3. Background of Codified Elastic Rules
- 4. Nonlinear design rules proposal
 - > 4.1 Scope
 - 4.2 Definition / Terms related to analysis
 - ➤ 4.3 Analysis Methods: limit load & elastic-platic analysis
 - ➤ 4.4 to 4.7 Plastic collapse, plastic instability, local failure, fatigue Ke, plastic shakedown, fatigue*
 - * for each of them: Analysis methods, Material properties, Criteria, Practical recommendations
 - 4.8 Quality management
 - 4.9 Stress classification & piping analysis
- 5. Learning from benchmarks from Part 3 of the Project
- √ 6. Conclusions
- ✓ Appendix A1: Draft International Harmonized Code Case
- ✓ Appendix A2: Cyclic plasticity models

4-0-

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Guidelines for Plastic Collapse: C_L (1/2)

Analysis Methods

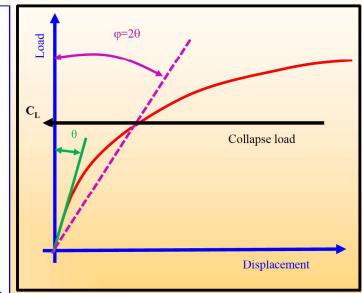
- √ Limit analysis: C_L
- ✓ Elastic plastic analysis:
 - "double slope" method

$$\circ \quad C_1 \text{ for } \varphi = 2 \theta$$

■ Alternative: $\varepsilon_{tot} \le 1\%$

Material properties

- √ For limit analysis
 - R_{p0.2,T} at max temperature in the transient
- √ For elastic-plastic analysis
 - Engineering stress-strain curves versus temperature

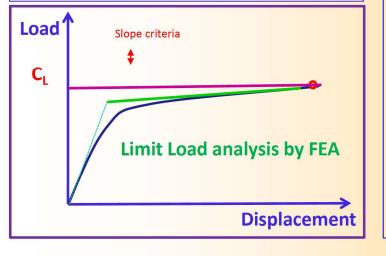




Guidelines for Plastic Collapse: C_L (2/2)

✓ Criteria

- ✓ For limit analysis or Elastic-plastic analyses: C_L
 - Level A(0) : $C_{app} \le 0.66 C_L$
 - Level B: $C_{app} \le 1.1 \times (0.66 C_L)$
 - Level C: $C_{app} \le 1.2 \times (0.66 C_L)$



✓ Practical recommendations

- ✓ For limit analysis by FEA
 - Initial geometry / Small displacement
 - "flat" stress-strain curve : E / 100
 - Isotropic hardening rule
 - load line displacement "min slope criteria" mesh refinement: not too much
 - multi-material: lower stress-strain curve
 - no functional requirement: gaps...
 - Von Mises (generally without correction to connect with Tresca criteria)
- ✓ Material properties: R_{p0.2,T}
- √ Sensitivity analysis
- ✓ Validation document and key references



Proposed rules by damage Guidelines for Plastic Instability: C₁ (1/1)

Limit load analysis

- No theoretical background for limit analysis; nevertheless it can be used with "precautions" and S_{flow} = (R_{p0.2%} + R_m)/2
- Same procedure than Collapse analysis, generally (!!!); C_L with R_{p0.2} proportional to C_L with S_{flow}
- Some components can be subjected to large displacement: proposal in the report

✓ Elastic-plastic analysis

- using "true stress-strain curve"
- large displacement has to be the reference method
- Evaluation of C_I for A% /k (around 5%)
- Criteria can be discussed material by material using (End Of Life ductility through maximum elongation A%)

Criteria

For limit Analysis and elastic-plastic Analysis

o Level A: Capp ≤ C_1 / 2.5 o Level B: Capp ≤ C_1 / 2.25 o Level C: Capp ≤ C_1 / 2.0 o Level D: Capp ≤ C_1 / 1.25

Note: Under certain conditions, (no large displacement), If limit load analyses were done, with limit load method: $S_{flow} / R_{p0.2} \ge 2.5/1.5 = 1.67$ no particular analyses required



(1)

Guidelines for Local Failure (1/1)

 $\sigma_{\parallel} + \sigma_{\parallel} + \sigma_{\parallel}$ by direct elastic-plastic analysis will be proposed with a dedicated criteria if the stress are issued from :

ightharpoonup Elastic analysis: $\sigma_{\parallel} + \sigma_{\parallel} + \sigma_{\parallel} \leq 4 \text{ S}$

Non-linear analysis: $\varepsilon_{peq} + \varepsilon_{cf} \le \varepsilon_{L}$ (2 to 7)

(see appendix to this presentation)

- Not for level D criteria (accidental event),
- Only for class 1 in nuclear
- Refer to ASME VIII Div 2 background



Guidelines for Crack Analysis (1/2)



- Methods based on $J_{appl} \le J_{\Delta a}$ and $dJ/da_{applied} < dJ/da_{material}$
 - Engineering approach based on Reference Stress and need plastic collapse load of "cracked components"
 - Direct J estimation by elastic-plastic stress analysis and J post-treatment modules,
- Reference stress
- → Similar analysis as plastic collapse on "un-cracked" components
- Direct J estimation
- → by FEA with dedicated recommendation



Guidelines for Crack Analysis (2/2)

□ FEM Analysis Recommendation

- Mesh recommendations
- Type of elements around the crack
- Size of elements at the crack tip
- Mesh of the remaining structure
- Analysis method
- Performing the analysis
 - Defining the material properties of the component part
 - Defining limit conditions
 - Defining load conditions
 - Performing the calculation
 - Verifying the calculation
 - Interpreting the results



Guidelines for Fatigue K_e - K_v (1/1)

✓ Definitions

$$K_e = \frac{1 + \nu}{1 + \nu^*} \frac{\Delta \varepsilon_{eq,VM}^t}{\Delta \varepsilon_{eq,VM}^e}$$

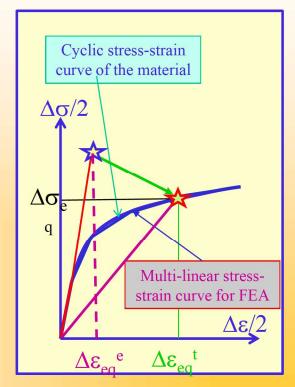
With

$$v^* = 0.5 - \frac{Es}{E}(0.5 - v)$$

✓ Method:

$$Es = \frac{2(1+v^*)}{3} \frac{\Delta \sigma_{eq,VM}}{\Delta \varepsilon_{eq,VM}^t}$$

- Elastic-plastic analysis: $\Delta \epsilon_{\rm eq}^{\rm e}$, $\Delta \sigma_{\rm eq}$, $\Delta \epsilon_{\rm eq}^{\rm t}$
- Multilinear kinematic hardening
- Used of cyclic stress-strain curve as stress-strain curve for FEA





Guidelines for Fatigue Direct cyclic strain amplitude evaluation (1/1)

- Direct cyclic elastic-plastic analysis
- Material constitutive equations
- Isotropic/ Kinematic/ Mixed
 - Multi-linear hardening
 - Chaboche models or similar
 - Others: Amstrong Frederick...

>NOT VALID EXISTING Model

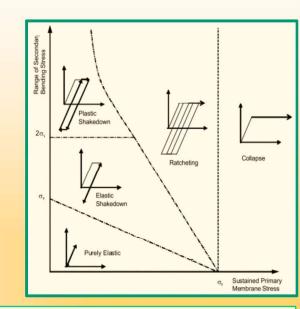
- Chaboche "modified" in Appendix A2
 - Cyclic variation of Young modulus E
 - Cyclic variation of Yield strength
 - Cyclic variation of NLKH: Cl and γI
 - Variation of cyclic load amplitude



Guidelines for plastic shakedown (1/1)

- √ 2 methods proposed:
 - Elastic-perfectly plastic stress-strain curve
 - Same constitutive equations similar than Fatigue:

"Model Chaboche evolutive method"



Detailed writing on-going

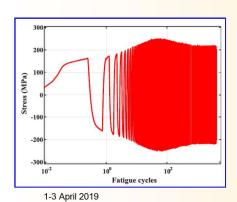


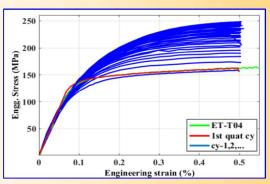
App. 2: NLK Cyclic Plasticity Constitutive equation 1/4

- Strongly material dependent
- ✓ For stainless steel, a step by step is in progress (mainly in ANL-USA)
 - Cold temperature creep negligible criteria

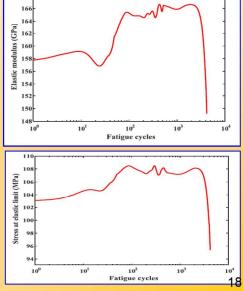
 Simulation of specimen monodim, fatigue tests: hardening, followed by softening, significant change of Young Modulus

and Yield stress versus cycles





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Experiment

Prediction (Tensile) Prediction (Half-life)

10-1

280

140

Stress (MPa)

-280

-420

10-2

10²

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NLK Cyclic Plasticity Constitutive equation (2/4)

Chaboche classical models (including Amstrong Frederic)

Does not fit experimental results

Based on ANL Chaboche
"evolutionary" model

Good fitting of experimental results

PVP2017-65890

100

Fatigue cycles

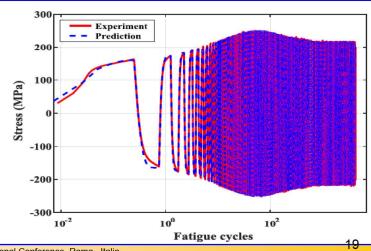
10¹

Prediction (Time/Cycle dependent)

Is it Possible to Get-rid of S-N Curve for Fatigue Evaluation?: A Fully Mechanistic Model of 316SS Reactor Steel for Fatigue Life Evaluation and **PVP2017-65876**

Fatigue Modeling of 508 LAS under Variable Amplitude Loading: A Mechanistic Based Analytical Approach

Bipul Barua-Subhasish Mohanty-William K. Soppet-Saurindranath Majumdar-Krishnamurti Natesan (ANL)



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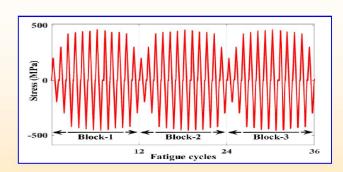
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Open points on NLK Cyclic Plasticity Models (3/4)

√ Variable amplitude



- ✓ Non-proportional loads
- Finally: real practical loads on industrial structures



→ Needs of more reference tests...

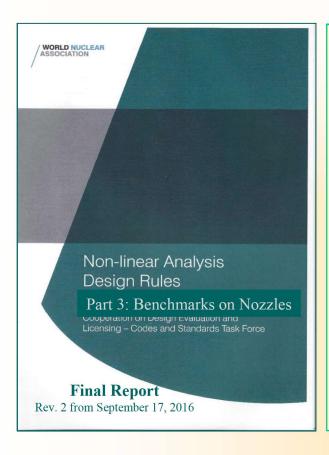
20



Synthesis on cyclic plasticity model (4/4)

- Basic Chaboche or Amstrong Frederic models are not running on selected laboratory tests
- ✓ Before fatigue cyclic analysis, any model, as NLK model, has to be validated on very close material against:
 - a standard strain control fatigue test
 - a variable amplitude fatigue test
 - a bi-axial fatigue test
 - a non-proportional fatigue test
- ✓ Specific Dedicated QA will be required:
 - Validation tests program
 - Computer Code validation
 - User qualification through dedicated training





Benchmarks

- 2 benchmarks to apply and improve "guidelines report"
 - Large LAS vessel nozzle under pressure and piping loads
 - Small SS piping nozzle under pressure and thermal loads
 - All the data are in the report
 - Result presentation are also in the report
- Bench 1: Large vessel reinforced nozzle (160" x 28") under monotonic P and piping loads (Ferrit Steel)
 - Elastic codified rules
 - Elastic-plastic
 - Limit loads
- Bench 2: Piping tee (3" x 28") under <u>cyclic</u> thermal shocks (stainless steel)
 - · Elastic codified rules
 - Simplified elastic-plastic and K_e in Fatigue
 - Direct cyclic and plastic shakedown



Other topics in the Project

- Project extension will cover:
 - Stress classification rules, like nozzle areas:
 - Ongoing with Benchmark 1: nozzle reinforcement rules (by applying existing Codified rules for stress classification)
 - High seismic loads: fatigue-ratchetting and strain criteria
 - Ongoing outside of this project in international ASME Working Group on Design Methodology
 - Elastic follow-up rules for piping systems
 - refer to ASME III new App. F
 - Later review and strain criteria

upper bound

large displacement



Open points

- ✓ Limit load:
 - using FEM
 - Criteria for "flat" stress-strain curve
 - Criteria for "flat" load-displacement curve
 - Large load for plastic instability
- Monotonic EP analysis
 - Monotonic stress-strain curves
- ✓ Cyclic stress-strain curves
 - Model selection: hardening/softening, Young Modulus, S_y variation, load variation
 using monotonic stress-strain curve and fatigue test for material calibration
 - Methods + Constitutive equation validation:
 - On simple specimen with different fatigue load histories
 - On simple "realistic structures"
- ✓ Environment: some cyclic tests results are different in air and in PWR ????
- ✓ Particular cases: piping systems and fittings
 - Elastic follow-up
 - Non-linear analysis of fittings and not on all the piping system
 - High plastic instability load
 - High seismic load
- ✓ Strain criteria: max or cumulative value
- ✓ More reference tests are needed...



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Conclusion and updated planning

Code Comparison task is finished

No existing Code propose a detail procedure with all the material properties needed, in particular for cyclic loads

Recommended practice Document

- OK: for plastic collapse, instability
- OK: for local failure
- OK: for crack rupture
- OK: for K_e non linear evaluation
- On-going: : for cyclic: fatigue and shakedown
- > Supplementary experimental and analysis program is needed
- > First EPERC report: Summer 2019
- Periodically revised associated to benchmark results

✓ Benchmarks

- Elastic approaches :
- Final Benchmark results:
- Dedicated Workshop:

August 2019

Final report and Draft Code Case:

End 2019

A dedicated EPERC TG on Non-Linear Analysis after this Conference...





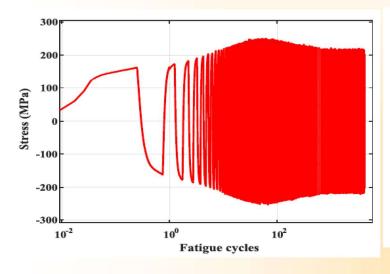
Thanks for your Attention !!!

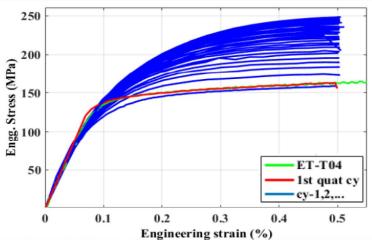


Open for questions and/or comments...



Based on Chaboche "evolutionary" model



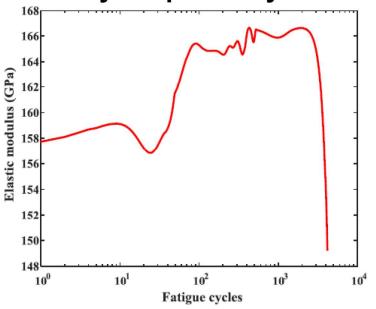


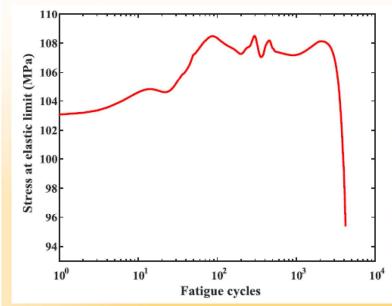
For 316 Stainless Steel



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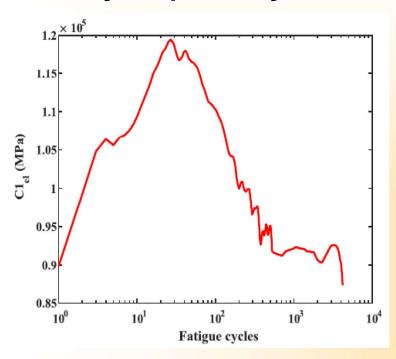
Cyclic plasticity models for Fatigue and Ratcheting

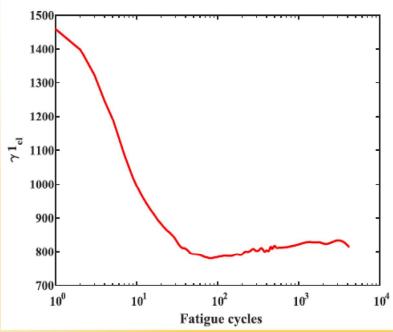




For 316 Stainless Steel







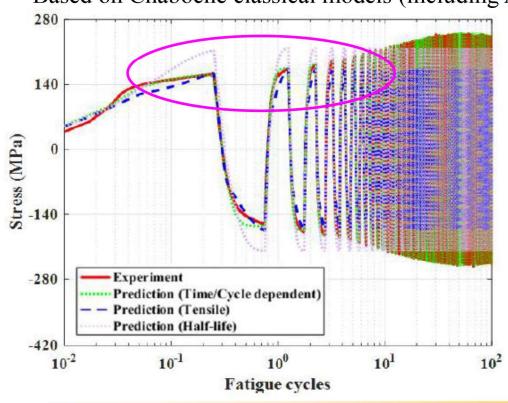
For 316 Stainless Steel

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Based on Chaboche classical models (including Amstrong Frederic)



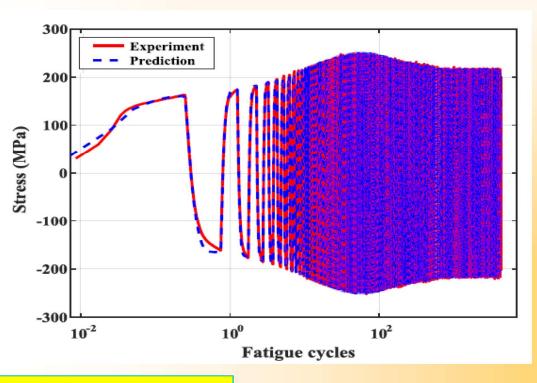
Simulated vs
experimental axial stress
history of specimen for
first 100 cycles.
Predictions are shown
from simulation using
time/cycle-dependent
parameters and two sets
of time-independent
parameters (estimated
from tensile test and
half-life cycle of fatigue
test, respectively)

For 316 Stainless Steel

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Based on Chaboche "evolutionary" model



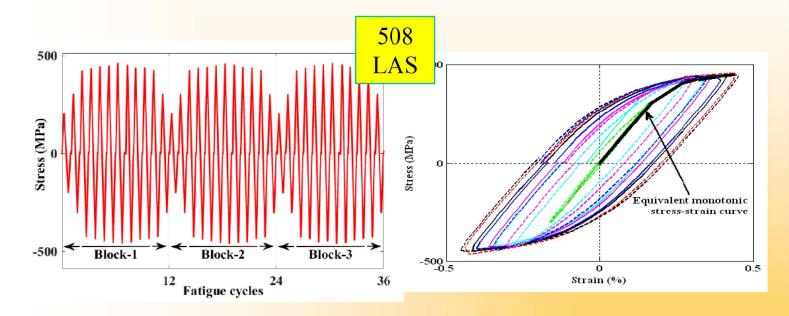
Simulated (using evolutionary cycle plasticity model) vs experimental axial stress history of specimen for whole fatigue life (4201 cycles)

For 316 Stainless Steel

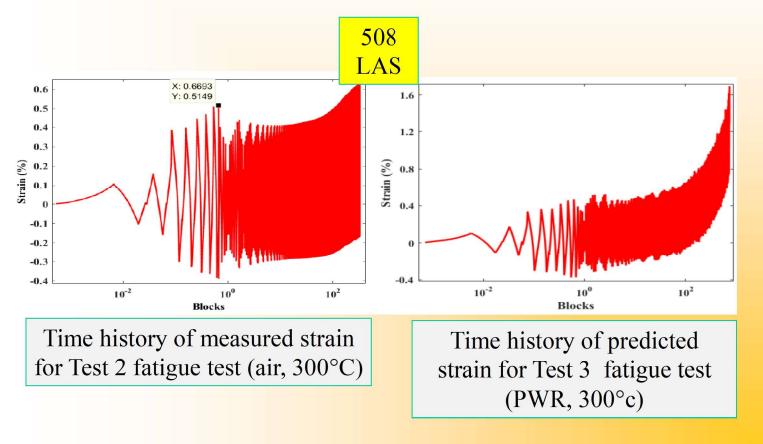
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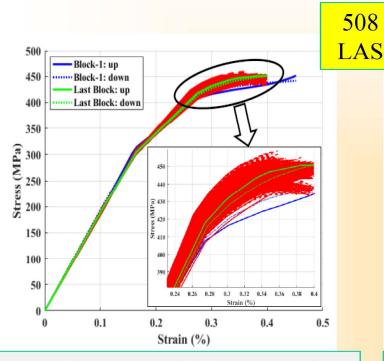












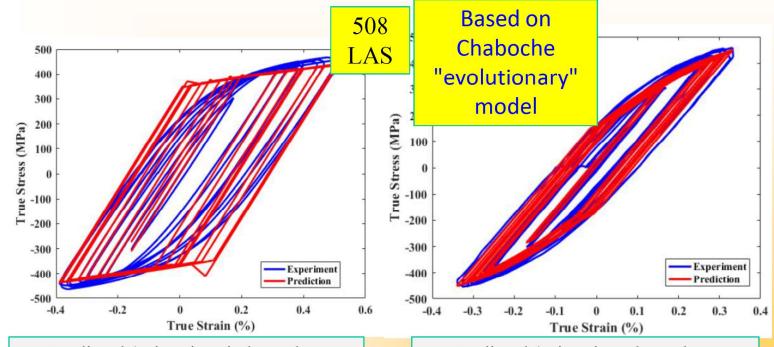
Block average equivalent monotonic strain-stress graphs (estimated from Test 2 data) for base metal fatigue tested in air at 300°C

500 Block-1: up 450 Block-1: down Last Block: up 400 Last Block: down 350 Stress (MPa) 300 250 200 150 100 50 0.05 0.35 0.4 Strain (%)

Block average equivalent monotonic strainstress graphs (estimated from Test 3 data) base metal fatigue tested in PWR environment at 300°C

_34





Predicted (using time-independent parameters from Test 1) versus experimental hysteresis curves for first block (with 12 variable-amplitude cycles) of Test 2

Predicted (using time-dependentamplitude-independent parameters) versus experimental hysteresis curves for 20th block (with 12 variable-amplitude cycles) of Test3



- Work continue on the subject at International level
- Experimental / Simulation of specimen and tests is needed before Code Developments